STANDARD IB CLUTCH IB314P0 • IB314P1

QUALITY IS STANDARD

- TAPERED ROLLER MAIN BEARINGS
- SEALED FOR LIFE PILOT BEARING
- VENTILATED CENTER PLATES AND DRIVE RING
- OPTIONAL SINTERED IRON PLATES
- BUILT IN HEX NUT
- MORE SUITABLE FOR SIDE LOAD APPLICATIONS
- LESS MAINTENANCE
- IMPROVED HEAT DISSIPATION
- · CREATES 25% HIGHER TORQUE CAPACITY
- EASES ADJUSTMENT VERIFICATION



SPECIFICATIONS - IB314P0, IB314P1

Model Number	SAE HSG.	Dimension "A" mm (in)	Max. Input Torque Nm (lb-ft)		Maximum Safe Speed				
			Organic	Sintered	Solid Plates		Split Plates		Weight kg (lbs)
					Cast Drive Ring	Nodular Drive Ring	Cast Drive Ring	Nodular Drive Ring	18 (133)
IB314P0	0	100 (3.9370)	3297 (2430)	4125 (3040)	N/A	2800	N/A	N/A	270 (595)
IB314P1	1	80 (3.1496)							

LOAD CLASSIFICATIONS BASED UPON AGMA LOAD CHARACTERISTICS

DDIME MOVED	DURATION	DRIVEN MACHINE LOAD CLASSIFICATIONS				
PRIME MOVER	OF SERVICE	UNIFORM	MODERATE SHOCK	HEAVY SHOCK		
Electric motor	Up to 3 hours per day	1.00	1.25	1.50		
	3-10 hours per day	1.00	1.25	1.75		
	Over 10 hours per day	1.25	1.50	2.00		
Multi-cylinder internal combustion engine	Up to 3 hours per day	1.00	1.25	1.75		
	3-10 hours per day	1.25	1.50	2.00		
	Over 10 hours per day	1.50	1.75	2.25		
Multi-cylinder internal	Up to 3 hours per day	1.50	1.75	2.25		
combustion engine	3-10 hours per day	1.75	2.00	2.50		
with high torque rise	Over 10 hours per day	2.00	2.25	2.75		
Single cylinder internal combustion engine	Up to 3 hours per day	1.25	1.50	2.00		
	3-10 hours per day	1.50	1.75	2.25		
	Over 10 hours per day	1.75	2.00	2.50		

All clutch engagements to be with prime mover below 1000 RPM. High inertia loads may require use of larger clutch. Contact Twin Disc application engineering department for assistance.

TO CALCULATE APPLICATION TORQUE:

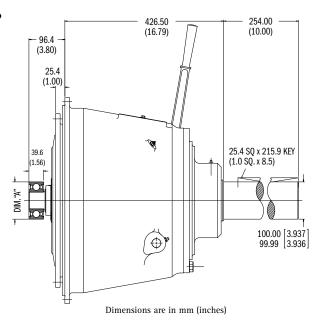
 $\frac{5252 \text{ x HP}}{\text{Engine RPM}} = \text{Torque}$

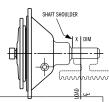
Torque x Load Factor = Application Torque

Use load factor from chart at left



IB314P





IB314P0 & IB314P1 - ALLOWABLE SIDE LOAD, KG (LBS)

PTO MODEL	RPM	X DISTANCE, mm (in) – see sketch							
		25.4 (1.0)	50.8 (2.0)	76.2 (3.0)	101.6 (4.0)	127.0 (5.0)	152.4 (6.0)	177.8 (7.0)	
IB314P0 M2731	1000	4068 (8969)	3881 (8557)	3711 (8182)	3555 (7838)	2420 (0070)	2758 (6080)	2471 (5448)	
	1200	3853 (8494)	3676 (8104)	3514 (7748)	3367 (7423)	3120 (6878)			
	1800	3412 (7522)	3255 (7176)	3112 (6862)	2982 (6574)	2862 (6309)			
	2400	3131 (6903)	2987 (6586)	2856 (6296)	2737 (6033)	2626 (5790)	2525 (5566)	2430 (5358)	
IB314P1	1000	4072 (8978)	3650 (8048)	3000 (6616)	2547 (5616)	2213 (4879)	1956 (4313)	1753 (3865)	
M2529	1200	3857 (8503)	3030 (0040)						
	1800	3415 (7530)	3260 (7186)						
	2400	3153 (6911)	2991 (6595)	2860 (6307)					

The following general formula should be used for determining the actual applied load: $L = \frac{126,000 \text{ x HP}}{\text{N x D}} \text{ x F x LF}$

WHERE L = Actual Applied Load (lbs)

N = Shaft Speed (RPM)

D = Pitch Diameter (in) of Sheave, etc.

F = Load Factor

1.0 for Chain or Gear Drive, 1.5 for Timing Belts, 2.5 for All V Belts, 3.5 for Flat Belts

LF = 2.1 for Reciprocating Compressors and other Severe Shock Drives and 1.8 for Large Inertia Type Drives (i.e. crushers, chippers, planers, etc.)

Compound Drives and Power Engaged Power Take-Off applications must have written factory review.

Twin Disc, Incorporated reminds users of these products that their safe operation depends on use in compliance with engineering information provided in our catalog. Users are also reminded that safe operation depends on proper installation, operation and routine maintenance and inspection under prevailing conditions. It is the responsibility of users (and not Twin Disc, Incorporated) to provide and install guards or safety devices which may be required by recognized safety standards or by the Occupational Safety and Health Act of 1970 and its subsequent provisions.

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